



URANS simulations of a horizontal axis wind turbine under stall condition using Reynolds stress turbulence models

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Supervisor: Prof. Segen F. Estefen July 2020

Outline:

1. Introduction

- 1.1. Problem Definition and Scope of the Present Work
- 1.2. Literature Review
- 1.3. Research Objectives

2. Methodology

- 2.1. Numerical Model
- 2.2. Wind Turbine Model and Computational Conditions
- 2.4. Computational Domain and Boundary Conditions and Grid
- 3. Verification
- 4. Results and Discussion
- 5. Conclusions and Future Works
- 6. Contributions



<u> </u>	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Evolution of the wind turbine size:



h			Introduction	Methodology	Verification	Results & Discussion	Conclusions
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1. Alexandra (1. Alexandra)	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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1. Annual 1.	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Evolution of the wind turbine size:



Modeling accuracy of the turbulent flow around the wind turbine rotors



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Modeling the turbulent flow around a wind turbine:

$$\frac{\partial(\rho U_i)}{\partial t} + \frac{\partial(\rho U_j U_i)}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial(2\mu S_{ij})}{\partial x_j}$$



be		Introduction	Methodology	Verification	Results & Discussion	Conclusions
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CFD



<u> </u>	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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URANS

- 1. Reynolds stress turbulence models
- 2. Nonlinear quadratic and cubic turbulent eddy viscosity models
- 3. Linear turbulent eddy viscosity models



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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 Accuracy



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The choice of a proper turbulence model is a compromise between the accuracy and theC. Argyropoulos and N. Markatos (2015)computational effort.



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Introduction Methodology	Verification	Results & Discussion	Conclusions
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Correct prediction of the turbulent flow characteristics around

wind turbines operating under stall condition



Scope of the Present Work:

Evaluation of the capabilities of five URANS-based turbulence models to predict the turbulent flow characteristics around a horizontal axis wind turbine operating under stall condition.



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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The use of nonlinear eddy viscosity and Reynolds stress turbulence models

S. A. Abdulqadir et al. (2017)



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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The use of nonlinear eddy viscosity and Reynolds stress turbulence models

S. A. Abdulqadir et al. (2017)

F. Carneiro (2019), H. Wang et al. (2019), H. Rahimi et al. (2018),
A. Ebrahimi and M. Movahhedi (2018), M. Moshfeghi et al. (2017), H. Rahimi et al. (2016),
M. Nobari et al. (2016), M. Make and G. Vaz (2015), S. Guntur and N. N. Sørensen (2015),
L. Daroczy (2015), J. Y. You et al. (2013), Q. Wang et al. (2012), G. Yu et al. (2011),
N. Tachos (2010), A. Spentzos et al. (2007), N. N. Sørensen et al. (2002), H. Snel (1998, 2003),

linear $k - \omega$ SST



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Discussion	М		Introduction	Methodology	Verification	Results & Discussion	Conclusion
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The use of nonlinear eddy viscosity and Reynolds stress turbulence models

S. A. Abdulqadir et al. (2017)

- 1. The latest formulations of the turbulence models are not considered.
- 2. Lack of detailed information about the computational cost of each turbulence model.
- 3. It neglects the influence of the tower.



Introduction	Methodology	Verification	Results & Discussion	Conclusions
			DISCUSSION	

Research Objectives:

- Extending the knowledge of the turbulent flow characteristics around horizontal axis wind turbines.
- Assessing the performance of five URANS turbulence models, principally Reynolds stress models, in predicting the wind turbines aerodynamics under stall condition.



Introduction Methodology	Verification	Results & Discussion	Conclusions
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Introduction	Methodology	Verification	Results &	Conclusions
			Discussion	

- □ STARCCM+ is used.
- □ Modeling the fluid flow:

$$\frac{\partial(\rho U_i)}{\partial t} + \frac{\partial(\rho U_j U_i)}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial(2\mu S_{ij} + \tau_{ij})}{\partial x_j},$$

$$\frac{\partial U_i}{\partial x_i} = 0,$$



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□ Five turbulence models are used:

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- □ Five turbulence models are used:
- 1. Reynolds stress turbulence model
- 1.1. Linear pressure-strain model (LRST)
- 1.2. Quadratic pressure-strain model (QRST)
- 1.3. Elliptic blending pressure-strain model (ERST)
- 2. SST k-w turbulence model
- 2.1. Linear eddy viscosity SST k-w
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with continuity:

$$\frac{\partial U_i}{\partial x_i} = 0,$$

$$\tau_{ij} = 2\mu_t S_{ij} - \frac{2}{3}\rho k \delta_{ij},$$

Boussinesq's hypothesis

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$$\tau_{ij} = 2\mu_t S_{ij} - \frac{2}{3}\rho k \delta_{ij},$$

An alignment between the Reynolds stresses and the mean deformation rate of the flow.

$$\gamma_{\tau} = \operatorname{atan}\left\{\frac{\tau_{23}}{\tau_{12}}\right\}, \qquad \gamma_{g} = \operatorname{atan}\left\{\frac{\frac{\partial U_{2}}{\partial x_{3}} + \frac{\partial U_{3}}{\partial x_{2}}}{\frac{\partial U_{1}}{\partial x_{2}} + \frac{\partial U_{2}}{\partial x_{1}}}\right\},$$

F. R. Menter (1994), J. Boussinesq (1877)



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with continuity:

$$\frac{\partial U_i}{\partial x_i} = 0,$$

$$\tau_{ij} = 2\mu_t S_{ij} - \frac{2}{3}\rho k \delta_{ij} - 2 \times 0.04645 \mu_t \left(O_{ij} \cdot S_{ij} - S_{ij} \cdot O_{ij} \right),$$

$$O_{ij} = \frac{\Omega_{ij}}{\sqrt{(S_{ij} - \Omega_{ij})(S_{ij} - \Omega_{ij})}},$$

F. R. Menter (1994), J. Boussinesq (1877). P. Durbin (1996), S. K. Arolla, P. A. Durbin (2013)



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$$\begin{split} \frac{\partial(\tau_{ij})}{\partial t} + \nabla \cdot \left(\tau_{ij} \cdot U\right) &= \nabla \cdot \left[\mu + \frac{\mu_t}{\sigma_k}\right] \frac{\nabla \tau_{ij}}{\rho} + P_{ij} - \frac{2}{3} \varepsilon \mathbf{I} + \Pi_{ij}, \\ \frac{\partial(\rho \varepsilon)}{\partial t} + \nabla \cdot (\rho \varepsilon U) &= \\ \\ \text{STARCCM} + (2017) & \nabla \cdot \left[\mu + \mu_t \sigma_k\right] \nabla \varepsilon + \frac{\varepsilon}{k} \left[C_{\varepsilon 1} \left(\frac{1}{2} tr(P_{ij})\right) - C_{\varepsilon 2} \rho \varepsilon\right], \end{split}$$

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$$\frac{\partial(\tau_{ij})}{\partial t} + \nabla \cdot \left(\tau_{ij} \cdot U\right) = \nabla \cdot \left[\mu + \frac{\mu_t}{\sigma_k}\right] \frac{\nabla \tau_{ij}}{\rho} + P_{ij} - \frac{2}{3} \varepsilon \mathbf{I} + \Pi_{ij}, \quad \text{turbulent pressure-strain interaction} \\ \frac{\partial(\rho \varepsilon)}{\partial t} + \nabla \cdot (\rho \varepsilon U) = \\ \\ \text{STARCCM} + (2017) \\$$

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 $\Pi = \Pi_s + \Pi_r + \Pi_w.$

STARCCM+ (2017), B. E. Launder, N. Shima (1989)



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STARCCM+ (2017), B. E. Launder, N. Shima (1989), C. G. Speziale et al. (1991)

GERO



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$$\begin{aligned} \Pi_{ij} &= -\left[C_{s1}\rho\varepsilon + C_{r4}tr(P)\right]A_{ij} \\ &+ C_{s2}\rho\varepsilon \left(A_{ik}A_{kj} - \frac{1}{3}A_{mn}A_{mn}\delta_{ij}\right) + \\ &\left(C_{r3} - C_{r3}^*\sqrt{A_{ij}A_{ij}}\right)\rho kS_{ij} + \\ &C_{r1}\rho k \left(A_{ik}S_{jk} + S_{ik}A_{jk} - \frac{2}{3}A_{mn}S_{mn}\delta_{ij}\right) \\ &+ C_{r2}\rho k \left(A_{ik}\Omega_{jk} + \Omega_{ik}A_{jk}\right), \end{aligned}$$

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Anisotropy Tensor

STARCCM+ (2017), B. E. Launder, N. Shima (1989), C. G. Speziale et al. (1991)

GERO

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$$\Pi - \varepsilon = \left(1 - \alpha^3\right) (\Pi^w - \varepsilon^w) + \alpha^3 \left(\Pi^h - \varepsilon^h\right)$$

blending parameter

STARCCM+ (2017), B. E. Launder, N. Shima (1989), C. G. Speziale et al. (1991), P. Durbin, (1993), S. Lardeau, R. Manceau (2014)



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2.2
$$\Pi^{h}_{ij} = -[C_{1}\rho\varepsilon + C_{1s}tr(P)]A_{ij}$$
$$+ (C_{3} - C_{3s}\sqrt{A_{ij}A_{ij}})\rho kS_{ij}$$
$$+ C_{4}\rho k \left(A_{ik}S_{jk} + S_{ik}A_{jk} - \frac{2}{3}A_{mn}S_{mn}\delta_{ij}\right) + C_{5}\rho k \left(A_{ik}\Omega_{jk} + \Omega_{ik}A_{jk}\right).$$

$$\frac{\partial(\tau_{ij})}{\partial t} + \nabla \cdot \left(\tau_{ij} \cdot U\right) = \nabla \cdot \left[\mu + \frac{\mu_t}{\sigma_k}\right] \frac{\nabla \tau_{ij}}{\rho} + P_{ij} - \frac{2}{3} \varepsilon \mathbf{I} + \prod_{ij}, \qquad C_5 \rho k \left(A_{ik} \Omega_{jk} + \Omega_{ik} A_{jk}\right).$$

$$\Pi - \varepsilon = \left(1 - \alpha^3\right) (\Pi^w - \varepsilon^w) + \alpha^3 \left(\Pi^h - \varepsilon^h\right)$$

$$\Pi^w_{ij} = 5 \frac{\varepsilon}{k} \left[\frac{\tau_{ik}}{\rho} n_j n_k + \frac{\tau_{jk}}{\rho} n_i n_k - \frac{1}{2} \frac{\tau_{kl}}{\rho} n_k n_l \left(n_i n_j + \delta_{ij}\right)\right]$$

2

STARCCM+ (2017), B. E. Launder, N. Shima (1989), C. G. Speziale et al. (1991), P. Durbin, (1993), S. Lardeau, R. Manceau (2014)

Introduction	Methodology	Verification	Results & Discussion	Conclusions
			DISCUSSION	

- □ STARCCM+ is used.
- Modeling the fluid flow:

$$\frac{\partial(\rho U_i)}{\partial t} + \frac{\partial(\rho U_j U_i)}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial(2\mu S_{ij} + \tau_{ij})}{\partial x_j},$$

$$\frac{\partial U_i}{\partial x_i} = 0,$$

- Discretization schemes:
 - Second-order schemes for convective and diffusion terms.
 - Second-order implicit scheme for transient term.
- A SIMPLE-type solver is used for pressure-velocity coupling
- □ The simulations are performed for four rotor revolutions.


Geometry and Computational Conditions:

□ The NREL phase VI horizontal axis wind turbine





Computational Conditions:

- 1. Four wind speeds: 5, 10, 15 and 25 m/s
- 2. Rotor angular velocity: 72 rpm
- 3. Turbulence intensity: 0.5%



Computational Domain and Boundary Conditions:



N	Introduction	Methodology	Verification	Results & Discussion	Conclusions
Grid:					













14 M	Introduction	Methodology	Verification	Results & Discussion	Conclusions
Grid:					





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Richardson (1911), Richards (1997), Stern et al. (2006), Mchale and Friedman (2009).



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Wind Speed: 25 m/s



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Grid	Grid Refinement	Time step (s)	# of cells
Ι	Fine	0.001155/1.2	26,806,032
Π	Medium	0.001155	17, 345, 028
III	Coarse	0.001155×1.2	11, 362, 711



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Grid	Power (W)	Thrust (N)
Ι	12574.13	3927.126
II	12108.03	3912.806
III	11150.63	3888.066



2 C	Introduction	Methodology	Verification	Results & Discussion	Conclusions		
Grid convergence study: $R_G = \frac{S_I - S_{II}}{S_{II} - S_{III}}$, $U_G = 1.25 \left \frac{S_I - S_{II}}{S_{II} - S_{III}} \right $.							
Wind Spood: 25 m/	-			$S_{II}(r_{12} - 1)$	7		

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Grid	Power (W)	Thrust (N)
Ι	12574.13	3927.126
II	12108.03	3912.806
III	11150.63	3888.066

Quantity	R_G	p	$U_G \% S_{II}$
Power	0.49	4.99	4.52
Thrust	0.58	4.03	0.58



	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Thrust and Power





Radial distribution of normal force coefficient



Chordwise pressure coefficient distribution

Wind speed 5 m/s

















0.8













<u> </u>	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Evaluation of the turbulent flow anisotropy

Boussinesq hypothesis

$$\gamma_{\tau} = \operatorname{atan}\left\{\frac{\tau_{23}}{\tau_{12}}\right\}, \quad \gamma_{g} = \operatorname{atan}\left\{\frac{\frac{\partial U_{2}}{\partial x_{3}} + \frac{\partial U_{3}}{\partial x_{2}}}{\frac{\partial U_{1}}{\partial x_{2}} + \frac{\partial U_{2}}{\partial x_{1}}}\right\},$$



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Evaluation of the turbulent flow anisotropy

Boussinesq hypothesis

$$\gamma_{\tau} = \operatorname{atan}\left\{\frac{\tau_{23}}{\tau_{12}}\right\}, \quad \gamma_{g} = \operatorname{atan}\left\{\frac{\frac{\partial U_{2}}{\partial x_{3}} + \frac{\partial U_{3}}{\partial x_{2}}}{\frac{\partial U_{1}}{\partial x_{2}} + \frac{\partial U_{2}}{\partial x_{1}}}\right\},$$

$$|\gamma_g - \gamma_\tau|$$



<u>1</u>	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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$$|\gamma_g - \gamma_\tau|$$

This evaluation is restricted to the boundary layer region













26		Introduction	Methodology	Verification	Results & Discussion	Conclusions
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The pressure at blade trailing edge at r/R = 0.80





The pressure at blade trailing edge at r/R = 0.80

The thrust

Wind speed 25 m/s





	Introduction	Methodology	Verification	Results & Discussion	Conclusions
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Computational cost analysis

The computations are performed on a desktop computer consisting of 64-bit Intel Processors i9-9900KF@ 3.60GHz and 64 GB RAM.



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Introduction	Methodology	Verification	Results &	Conclusions
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Introduction	Methodology	Verification	Results &	Conclusions
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For the wind speed 10 m/s, the turbulence models return different predictions for the extent of the separated flow region over the suction side.



Introduction	Methodology	Verification	Results &	Conclusions
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An evaluation of the Boussinesq hypothesis implies that the turbulence models employing this assumption are inadequate for the prediction of wind turbines aerodynamics.


Introduction	Methodology	Verification	Results &	Conclusions
	Methodology	verneation	Discussion	0011010310113

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- All the models resolve satisfactorily the fluctuation spectrum of the pressure at a single point.



	Introduction	Methodology	Verification	Results &	Conclusions
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- An evaluation of the Boussinesq hypothesis implies that the turbulence models employing this assumption are inadequate for the prediction of wind turbines aerodynamics.
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- Regarding the thrust force, all the models underestimate the lowfrequency fluctuations and also fail to predict the high-frequency fluctuations.



Introduction	Methodology	Verification	Results &	Conclusions
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- An evaluation of the Boussinesq hypothesis implies that the turbulence models employing this assumption are inadequate for the prediction of wind turbines aerodynamics.
- All the models resolve satisfactorily the fluctuation spectrum of the pressure at a single point.
- Regarding the thrust force, all the models underestimate the lowfrequency fluctuations and also fail to predict the high-frequency fluctuations.
- Finally, the ERST model appears to provide the best trade-off between the accuracy and the computational cost.



Future Work Fluid-structure interaction analysis





Future Work

- Simulações Aerodinâmicas
- Turbina eólica
- Simulações Hidrodinâmicas
- semi-submersível + amarração
- Simulações Hidro-aerodinâmicas

Turbina eólica + semi-submersível + amarração



Numero mínimo de células + tamanho do passo de tempo +

um modelo de turbulência adequado (Modelos não lineares ou modelos de Reynolds)

77

Future Work

Evaluation of the performance of the ERST and LRST models without using the wall function.







Contributions

Submitted Publications

Amiri M.M., Shadman M., Segen F. Estefen, URANS simulations of a horizontal axis wind turbine under stall condition using Reynolds stress turbulence models, Energy, 2020.



Questions?